

## STATICS – COMPRESSION MEMBERS – TUTORIAL 1

### SELF ASSESSMENT EXERCISE No.1

Find the radius of gyration and the slenderness ratio of a strut made from 5 m length of hollow tube 50 mm outer diameter and 40 mm inner diameter.

$$I = \pi(50^4)/64 - \pi(40^4)/64 = 181132 \text{ mm}^4$$

$$A = \pi(50^2)/4 - \pi(40^2)/4 = 706.8 \text{ mm}^2$$

$$k = \sqrt{\frac{I}{A}} = 16 \text{ mm} \quad L = 5000 \text{ mm} \quad \text{Slenderness Ratio} = 5000/16 = 312.3 \text{ mm}$$

### SELF ASSESSMENT EXERCISE No. 2

1. A steel strut is 0.15 m diameter and 12 m long. It is built in rigidly at the bottom but completely unrestrained at the top.

Calculate the buckling load taking  $E = 205 \text{ GPa}$ .

$$\text{MODE } n = \frac{1}{2} \quad I = \pi D^4/64 = \pi(0.15^4)/64 = 24.85 \times 10^{-6}$$

$$F_e = n^2 \pi^2 EI/L^2 = \frac{1}{2}^2 \pi^2 205 \times 10^9 \times 24.85 \times 10^{-6}/12^2 = 89.4 \times 10^3 \text{ N}$$

2. A steel strut has a solid circular cross section and is 8 m long. It is pinned at the top and bottom but unable to move laterally at the ends. The strut collapses under a load of 200 kN. Taking  $E = 205 \text{ GPa}$  calculate the diameter of the strut.

MODE  $n = 1$

$$F_e = 200\,000 = \pi^2 EI/L^2 = \pi^2 205 \times 10^9 \times I/8^2$$

$$I = 200\,000 \times 8^2 / \pi^2 205 \times 10^9 = 6.326 \times 10^{-6}$$

$$I = 6.326 \times 10^{-6} = \pi D^4/64 \quad \text{hence } D = 0.1065 \text{ m}$$

3. A shaft is made from alloy tubing 50 mm outer diameter and 30 mm inner diameter. The shaft is placed between bearings 3 m apart so that the ends are constrained to remain horizontal. The shaft also has to take a horizontal axial load. Taking  $E = 120 \text{ GPa}$  determine the maximum axial load before buckling occurs.

MODE  $n = 2$

$$I = \pi(50^4)/64 - \pi(30^4)/64 = 267035 \text{ mm}^4$$

$$F_e = n^2 \pi^2 EI/L^2 = 2^2 \pi^2 120 \times 10^9 \times 267 \times 10^{-9}/3^2 = 140 \times 10^3 \text{ N}$$

4. A strut is 0.2 m diameter and 15 m long. It is pinned at both ends. Calculate Euler's critical load.  
 $E = 205 \text{ GPa}$

MODE  $n = 1$

$$I = \pi(0.2^4)/64 = 78.54 \times 10^{-6} \text{ m}^4$$

$$F_e = \pi^2 EI/L^2 = \pi^2 205 \times 10^9 \times 78.54 \times 10^{-6}/15^2 = 706 \times 10^3 \text{ N}$$

### SELF ASSESSMENT EXERCISE No. 3

1.a A uniform slender elastic column of length  $L$  is pin jointed at each end and subjected to an axial compression load  $P$ .

Show that the Euler crippling load occurs when  $P = \pi^2 EI/L^2$  where  $I$  is the relevant second moment of area of the column and  $E$  is the modulus of elasticity of the material. State any assumptions made.

b A straight steel rod 0.5 m long and 0.01 m diameters loaded axially until it buckles. Assuming that the ends of the rod are pin jointed, determine the Euler crippling load. Assume  $E = 206$  GPa.

**Part (a) solution is in the tutorial section 2.2 and constitutes the major part of the marks.**

$$\text{Mode } n = 1 \quad I = \pi \times 0.01^4 / 64 = 490.87 \times 10^{-12} \text{ m}^4$$

$$P = \pi^2 EI / L^2 = \pi^2 \times 206 \times 10^9 \times 490.87 \times 10^{-12} / 0.5^2 = 3992 \text{ N}$$

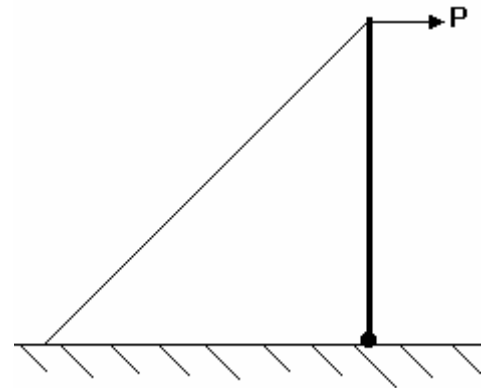
2.a The Euler buckling load  $P$  for a slender strut of length  $L$  and second moment of area  $I$ , pin jointed at each end, is given by

$$P = \pi^2 EI / L^2$$

$E$  is the modulus of elasticity of the material. Using this expression without proof, obtain the formula for the Euler buckling load when the strut is

- i. fixed (built in) at each end.
- ii. fixed at one end and pin jointed at the other.

b) A vertical pole 6 m long, pinned at the lower end and supported by a wire at the upper end. The pole consists of a tube 50 mm outside diameter and 40 mm inside diameter and the wire has an effective diameter of 6 mm. What is the maximum load  $P$  that this system can withstand before failure occurs?



For steel assume that the modulus of elasticity  $E$  is 206 GPa and for the wire assume that the ultimate stress is 480 MPa.

**Part (a) derivation is in the tutorial.**

Buckling load

$$F_e = \pi^2 EI / L^2$$

$$F_e = \pi^2 \times 206 \times 10^9 \times 181.1 \times 10^{-9} / 6^2$$

$$F_e = 10230 \times 10^3 \text{ N}$$

From triangle of forces

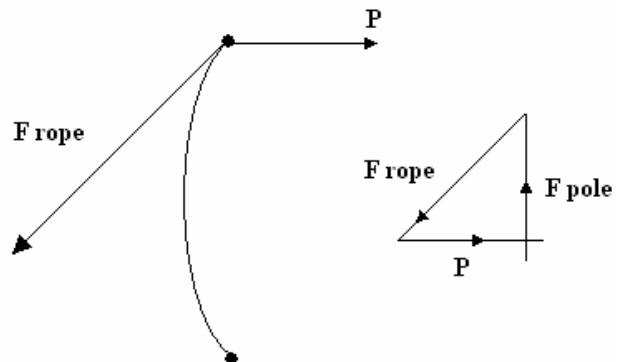
$$F_{\text{pole}} = P \quad F_{\text{rope}} = 0.707 P$$

Stress in rope = 480 MPa hence the breaking force is

$$F = \sigma A = 480 \times 10^6 \times \pi \times 0.006^2 / 4 = 13570 \text{ N}$$

$$\text{If the rope breaks } P = 13570 \times 0.707 = 9590 \text{ N}$$

Hence the rope breaks before the pole buckles.



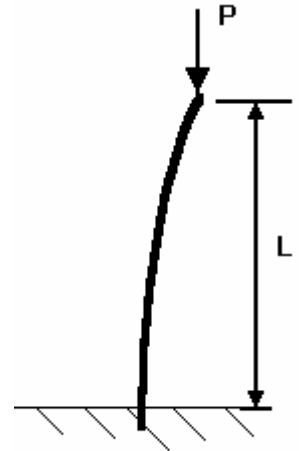
3.a A uniform slender strut of length  $L$  which is clamped at one end and free at the other is subjected to an axial compression load  $P$  as shown in fig.12. Show that according to EULER'S theory, the strut will buckle when  $P = (\pi/2L)^2 EI$  where  $I$  is the minimum second moment of area of the strut and  $E$  is the modulus of elasticity for the material.

b A straight steel rod 9 mm diameter is rigidly built into a foundation, the free end protruding 0.5 m normal to the foundation. An axial load is applied to the free end of the rod which deflects as shown in fig.12. Determine the following.

i. Euler's buckling load. **(636 N)**

ii. The deflection of the free end of the rod when the total compressive stress reaches the elastic limit. **(32.6 mm)**

For steel assume  $E = 200\text{GPa}$  and the stress at the elastic limit is  $300\text{MPa}$ .



**Part (a) derivation is in the tutorial.**

Mode  $n = 1/2$

$$I = \pi(0.009^4)/64 = 322 \times 10^{-12} \text{ m}^4$$

$$A = \pi(0.009^2)/4 = 63.6 \times 10^{-6} \text{ m}^2$$

Buckling load

$$P = n^2 \pi^2 EI/L^2 = (1/2)^2 \pi^2 200 \times 10^9 \times 322 \times 10^{-12} / 2^2 = 636 \text{ N}$$

$$\text{Stress } \sigma = 300 \times 10^6 = F/A + My/I$$

$$M = FX \quad y = 0.0045 \text{ m}$$

$$300 \times 10^6 = 636/63.6 \times 10^{-6} + 636X \times 0.0045/322 \times 10^{-12} = 10 \times 10^6 + 8.889 \times 10^9 X$$

$$290 \times 10^6 = 8.889 \times 10^9 X$$

$$X = 0.0326 \text{ m}$$