

FLUID MECHANICS D203
SAE SOLUTIONS TUTORIAL9 – COMPRESSIBLE FLOW

SELF ASSESSMENT EXERCISE No. 1

1. Calculate the specific entropy change when a perfect gas undergoes a reversible isothermal expansion from 500 kPa to 100 kPa. $R = 287 \text{ J/kg K}$.

$$T \text{ is constant so } \Delta s = mR \ln(p_1/p_2) = 1 \times 287 \times \ln(5/1) = 462 \text{ J/kg K}$$

2. Calculate the total entropy change when 2 kg of perfect gas is compressed reversibly and isothermally from 9 dm³ to 1 dm³. $R=300 \text{ J/kg K}$.

$$\Delta s = mR \ln(V_2/V_1) = 1 \times 300 \times \ln(1/9) = 470 \text{ J/kg K}$$

3. Calculate the change in entropy when 2.5 kg of perfect gas is heated from 20°C to 100°C at constant volume. Take $c_v = 780 \text{ J/kg K}$ (Answer 470 J/K)

$$\Delta s = m c_v \ln(T_2/T_1) = 2.5 \times 780 \times \ln(373/293) = 1318 \text{ J/kg K}$$

4. Calculate the total entropy change when 5 kg of gas is expanded at constant pressure from 30°C to 200°C. $R = 300 \text{ J/kg K}$ $c_v = 800 \text{ J/kg K}$ (Answer 2.45 kJ/K)

$$\Delta s = m c_p \ln(T_2/T_1) \quad c_p = R + c_v = 1100 \text{ J/kg K}$$

$$\Delta s = 5 \times 1100 \times \ln(473/303) = 2450 \text{ J/kg K}$$

5. Derive the formula for the specific change in entropy during a polytropic process using a constant volume process from (A) to (2).

$$s_2 - s_1 = (s_A - s_1) - (s_A - s_2)$$

$$s_2 - s_1 = (s_A - s_1) + (s_2 - s_A)$$

For the constant temperature process

$$(s_A - s_1) = R \ln(p_1/p_A)$$

For the constant volume process

$$(s_2 - s_A) = (c_v/R) \ln(T_2/T_A)$$

Hence

$$\Delta s = R \ln \frac{p_1}{p_A} + C_p \ln \frac{T_2}{T_A} + s_2 - s_1 \quad T_A = T_1$$

$$\text{Then } \Delta s = s_2 - s_1 = \Delta s = R \ln \left(\frac{p_1}{p_A} \right) + c_v \ln \left(\frac{T_2}{T_A} \right)$$

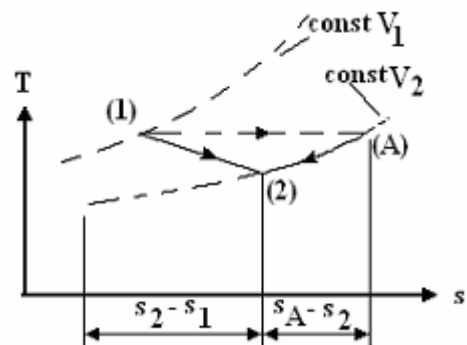
$$\text{Divide through by } R \quad \Delta s/R = \ln \left(\frac{p_1}{p_A} \right) + \frac{c_v}{R} \ln \left(\frac{T_2}{T_A} \right)$$

From the relationship between c_p , c_v , R and γ we have $c_p/R = \gamma / (\gamma - 1)$

From the gas laws we have $p_A/T_A = p_2/T_2$ $p_A = p_2 T_A / T_2 = p_2 T_1 / T_2$

Hence

$$\frac{\Delta s}{R} = \ln \left(\frac{p_1}{p_2} \right) + \frac{1}{\gamma - 1} \ln \left(\frac{T_2}{T_1} \right) = \ln \left(\frac{p_1}{p_2} \right) \left(\frac{T_2}{T_1} \right)^{\frac{1}{\gamma - 1}} = \ln \left(\frac{p_1}{p_2} \right) \left(\frac{T_2}{T_1} \right)^{\frac{\gamma}{\gamma - 1}}$$



6. A perfect gas is expanded from 5 bar to 1 bar by the law $pV^{1.6} = C$. The initial temperature is 200°C . Calculate the change in specific entropy.

$$R = 287 \text{ J/kg K} \quad \gamma = 1.4.$$

$$T_2 = T_1 \left(\frac{p_2}{p_1} \right)^{1-\gamma} = 473 \left(\frac{1}{5} \right)^{1-1.6} = 258.7 \text{ K}$$

$$\Delta s = R \ln \left(\frac{p_1}{p_2} \right) \left(\frac{T_2}{T_1} \right)^{\frac{\gamma}{\gamma-1}} = 287 \ln(5) \left(\frac{258.7}{473} \right)^{1.4} = -144 \text{ J/K}$$

7. A perfect gas is expanded reversibly and adiabatically from 5 bar to 1 bar by the law $pV^{\gamma} = C$. The initial temperature is 200°C . Calculate the change in specific entropy using the formula for a polytropic process. $R = 287 \text{ J/kg K}$ $\gamma = 1.4$.

$$T_2 = 473 \left(\frac{1}{5} \right)^{1/1.4} = 298.6 \text{ K}$$

$$\Delta s = R \ln \left(\frac{p_1}{p_2} \right) \left(\frac{T_2}{T_1} \right)^{\frac{\gamma}{\gamma-1}} = 287 \ln(5) \left(\frac{298.6}{473} \right)^{1.4} = 0$$

SELF ASSESSMENT EXERCISE No. 2

Take $\gamma = 1.4$ and $R = 283 \text{ J/kg K}$ in all the following questions.

1. An aeroplane flies at Mach 0.8 in air at 15° C and 100 kPa pressure. Calculate the stagnation pressure and temperature. (Answers 324.9 K and 152.4 kPa)

$$\frac{\Delta T}{T} = M^2 \frac{k-1}{2} = 0.8^2 \frac{1.4}{2} = 0.128 \quad \Delta T = 0.128 \times 288 = 36.86 \text{ K}$$

$$\frac{p_2}{p_1} = \left(M^2 \frac{k-1}{2} + 1 \right)^{\frac{k}{k-1}} = 1.128^{3.5} = 1.5243 \quad p_2 = 100 \times 1.5243 = 152.43 \text{ kPa}$$

2. Repeat problem 1 if the aeroplane flies at Mach 2.

$$\frac{\Delta T}{T} = M^2 \frac{k-1}{2} = 2^2 \frac{1.4}{2} = 0.8 \quad \Delta T = 0.8 \times 288 = 230.4 \text{ K}$$

$$T_2 = 288 + 230.4 = 518.4 \text{ K}$$

$$\frac{p_2}{p_1} = \left(M^2 \frac{k-1}{2} + 1 \right)^{\frac{k}{k-1}} = 1.8^{3.5} = 7.824 \quad p_2 = 100 \times 7.824 = 782.4 \text{ kPa}$$

3. The pressure on the leading edges of an aircraft is 4.52 kPa more than the surrounding atmosphere. The aeroplane flies at an altitude of $5\,000$ metres. Calculate the speed of the aeroplane. (Answer 109.186 m/s)

From fluids tables, find that $a = 320.5 \text{ m/s}$ $p_1 = 54.05 \text{ kPa}$ $\gamma = 1.4$

$$\frac{p_2}{p_1} = \frac{58.57}{54.05} = 1.0836 = \left(M^2 \frac{k-1}{2} + 1 \right)^{\frac{k}{k-1}}$$

$$1.0836 = \left(M^2 \frac{1.4-1}{2} + 1 \right)^{\frac{1.4}{1.4-1}} = (0.2 M^2 + 1)^{3.5}$$

$$1.0232 = 0.2 M^2 + 1 \quad M = 0.3407 = v/a \quad v = 109.2 \text{ m/s}$$

4. An air compressor delivers air with a stagnation temperature 5 K above the ambient temperature. Determine the velocity of the air. (Answer 100.2 m/s)

$$\frac{\Delta T}{T_1} = \frac{v_1^2(k-1)}{2\gamma RT_1} \quad \Delta T = \frac{v_1^2(1.4-1)}{2 \times 1.4 \times 287} = 5 \text{ K} \quad v_1 = 100.2 \text{ m/s}$$

SELF ASSESSMENT EXERCISE No. 3

1. A Venturi Meter must pass 300g/s of air. The inlet pressure is 2 bar and the inlet temperature is 120°C. Ignoring the inlet velocity, determine the throat area. Take C_D as 0.97.

Take $\gamma = 1.4$ and $R = 287 \text{ J/kg K}$ (assume choked flow)

$$m = C_d A_2 \sqrt{\left[\frac{2\gamma}{\gamma-1} \right] p_1 \rho_1 \left\{ (r_c)^{\frac{2}{\gamma}} - (r_c)^{1+\frac{1}{\gamma}} \right\}} \quad r_c = \left(\frac{2}{\gamma+1} \right)^{\frac{\gamma}{\gamma-1}} = \left(\frac{2}{2.4} \right)^{3.5} = 0.528$$

$$\rho_1 = p_1 / RT_1 = 2 \times 10^5 / (287 \times 393) = 1.773 \text{ kg/m}^3$$

$$0.3 = 0.97 A_2 \sqrt{7 \times 2 \times 10^5 \times 1.773 \left\{ (0.528)^{1.428} - (0.528)^{1.714} \right\}} = 0.97 A_2 \sqrt{166307}$$

$$A_2 = 758 \times 10^{-6} \text{ m}^2 \text{ and the diameter} = 31.07 \text{ mm}$$

2. Repeat problem 1 given that the inlet is 60 mm diameter and the inlet velocity must not be neglected.

$$m = C_d A_2 \sqrt{\frac{\left[\frac{2\gamma}{\gamma-1} \right] p_1 \rho_1 \left\{ (r_c)^{\frac{2}{\gamma}} - (r_c)^{1+\frac{1}{\gamma}} \right\}}{1 - \left(\frac{A_2}{A_1} \right)^2 \left(\frac{p_2}{p_1} \right)^{2/\gamma}}} \quad 0.3 = C_d A_2 \sqrt{\frac{166307}{1 - \left(\frac{A_2}{A_1} \right)^2}} \quad (0.4)$$

$$1 - (A_2/A_1)^2 \times 0.4 = 1738882 A_2^2$$

$$A_1^2 = (\pi \times 0.06^2 / 4)^2 = 7.99 \times 10^{-6} \text{ m}^2$$

$$1 - 50062 A_2^2 = 1738882 A_2^2$$

$$A_2^2 = 559 \times 10^{-9} \quad A_2 = 747.6 \times 10^{-6} \text{ m}^2$$

The diameter is 30.8 mm. Neglecting the inlet velocity made very little difference.

3. A nozzle must pass 0.5 kg/s of steam with inlet conditions of 10 bar and 400°C. Calculate the throat diameter that causes choking at this condition. The density of the steam at inlet is 3.263 kg/m³. Take γ for steam as 1.3 and C_D as 0.98.

$$m = C_d A_2 \sqrt{\left[\frac{2\gamma}{\gamma-1} \right] p_1 \rho_1 \left\{ (r_c)^{\frac{2}{\gamma}} - (r_c)^{1+\frac{1}{\gamma}} \right\}} \quad r_c = \left(\frac{2}{\gamma+1} \right)^{\frac{\gamma}{\gamma-1}} = \left(\frac{2}{2.3} \right)^{4.33} = 0.5457$$

$$0.5 = 0.98 A_2 \sqrt{8.667 \times 3.2626 \times 10 \times 10^5 \left\{ (0.5457)^{1.538} - (0.5457)^{1.538} \right\}} = 0.98 A_2 \sqrt{1.4526 \times 10^6}$$

$$A_2 = 423 \times 10^{-6} \text{ m}^2 \text{ and the diameter} = 23.2 \text{ mm}$$

4. A Venturi Meter has a throat area of 500 mm². Steam flows through it, and the inlet pressure is 7 bar and the throat pressure is 5 bar. The inlet temperature is 400°C. Calculate the flow rate. The density of the steam at inlet is 2.274 kg/m³. Take $\gamma = 1.3$. $R = 462 \text{ J/kg K}$. $C_d = 0.97$. From the steam tables $v_1 = 0.4397 \text{ m}^3/\text{kg}$ so $\rho_1 = 1/0.4397 = 2.274 \text{ kg/m}^3$

$$m = C_d A_2 \sqrt{\left[\frac{2\gamma}{\gamma-1} \right] p_1 \rho_1 \left\{ (r_c)^{\frac{2}{\gamma}} - (r_c)^{1+\frac{1}{\gamma}} \right\}}$$

$$m = 0.97 \times 500 \times 10^{-6} \sqrt{\left[\frac{2 \times 1.3}{1.3-1} \right] 7 \times 10^5 \times 2.274 \left\{ (5/7)^{1.538} - (5/7)^{1.764} \right\}}$$

$$m = 485 \times 10^{-6} \times 783 \quad m = 0.379 \text{ kg/s}$$

5. A pitot tube is pointed into an air stream which has an ambient pressure of 100 kPa and temperature of 20°C. The pressure rise measured is 23 kPa. Calculate the air velocity. Take $\gamma = 1.4$ and $R = 287 \text{ J/kg K}$.

$$\frac{p_2}{p_1} = \frac{123}{100} = 1.23 = \left(M^2 \frac{\gamma - 1}{2} + 1 \right)^{\frac{\gamma}{\gamma - 1}} \quad 1.23 = (0.2M^2 + 1)^{3.5}$$

$$1.0609 = 0.2M^2 + 1 \quad 0.0609 = 0.2M^2 \quad M = 0.5519$$

$$a = \gamma RT^{1/2} = (1.4 \times 287 \times 293)^{1/2} = 343.1 \text{ m/s}$$

$$v = 0.5519 \times 343.1 = 189.4 \text{ m/s}$$

6. A fast moving stream of gas has a temperature of 25°C. A thermometer is placed into it in front of a small barrier to record the stagnation temperature. The stagnation temperature is 28°C. Calculate the velocity of the gas. Take $\gamma = 1.5$ and $R = 300 \text{ J/kg K}$. (Answer 73.5 m/s)

$$\Delta T = 3 \text{ K} \quad \Delta T/T_1 = v^2/\gamma RT \quad c_p = \gamma R/(\gamma - 1)$$

$$\Delta T = 3 = v^2/2 c_p = v^2(\gamma - 1)/(2 \gamma R) = v^2(1.5 - 1)/(2 \times 1.5 \times 300)$$

$$v^2 = 5400 \quad v = 73.48 \text{ m/s}$$