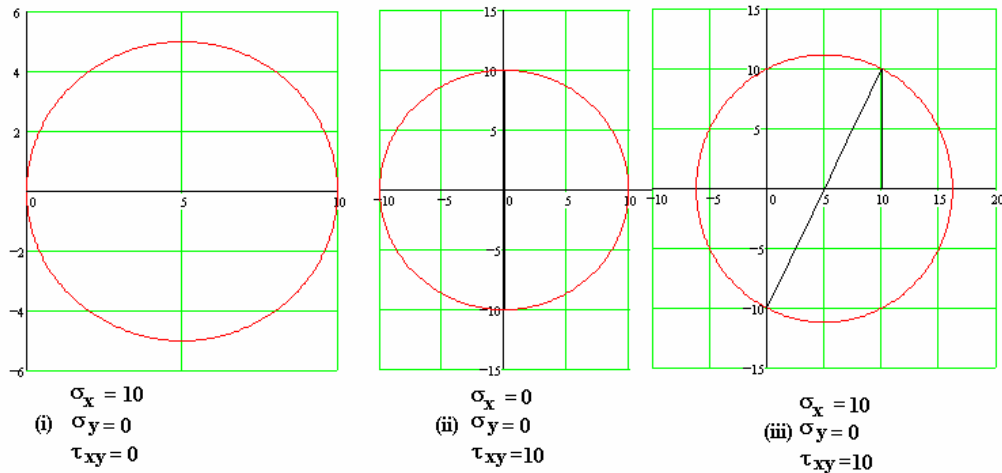


1.a Sketch Mohr's circle for a dimensional stress system in which:

- (i) $\sigma_x = 10$ $\sigma_y = 0$ $\tau_{xy} = 0$
- (ii) $\sigma_x = 10$ $\sigma_y = 0$ $\tau_{xy} = 10$
- (iii) $\sigma_x = 10$ $\sigma_y = 0$ $\tau_{xy} = 10$



1.b A hollow shaft is 40mm outer diameter and 20 mm inner diameter. It is subjected to a bending moment of 1 000 Nm. It also transmits a torque. Calculate the maximum torque allowed if

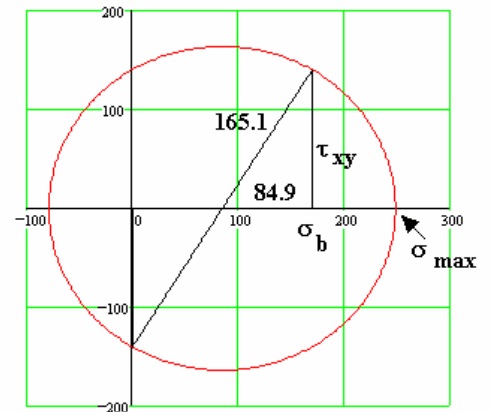
- (i) the maximum direct stress is 250 MPa.
- (ii) the maximum shear stress is 125 MPa.

$$D = 0.04 \text{ m} \quad d = 0.02 \text{ m} \quad I := \frac{\pi \cdot (D^4 - d^4)}{64} \quad I = 117.9 \times 10^{-9} \text{ m}^4$$

BENDING STRESS $M = 1000 \text{ kNm} \quad \sigma_b = \pm My/I \quad y = D/2 = 0.02 \text{ m} \quad \sigma_b = \pm 169.8 \text{ MPa}$

For torsion $J := \frac{\pi \cdot (D^4 - d^4)}{32} \quad J = 235.6 \times 10^{-9} \text{ m}^4$

- (i) At a point on the surface where the bending stress is 169.8 MPa with a maximum direct stress of 250 MPa Mohr's circle is as shown. From this the shear stress is $\tau_{xy} = \sqrt{(165.1^2 - 84.9^2)} = 141.6 \text{ MPa}$



From the torsion equation Torque = $T = \tau J/R$ $R = D/2 = 0.02 \text{ m}$

$$T = (141.6 \times 10^6 \times 235.6 \times 10^{-9}) / 0.02 = 1.668 \text{ kN m}$$

- (ii) The radius of the circle is the max shear stress so drawing the circle we get the following.

$$\tau_{xy} = \sqrt{(125^2 - 84.9^2)} = 91.74 \text{ MPa}$$

From the torsion equation Torque = $T = \tau J/R$ $R = D/2 = 0.02 \text{ m}$

$$T = (91.74 \times 10^6 \times 235.6 \times 10^{-9}) / 0.02 = 1.08 \text{ kN m}$$

